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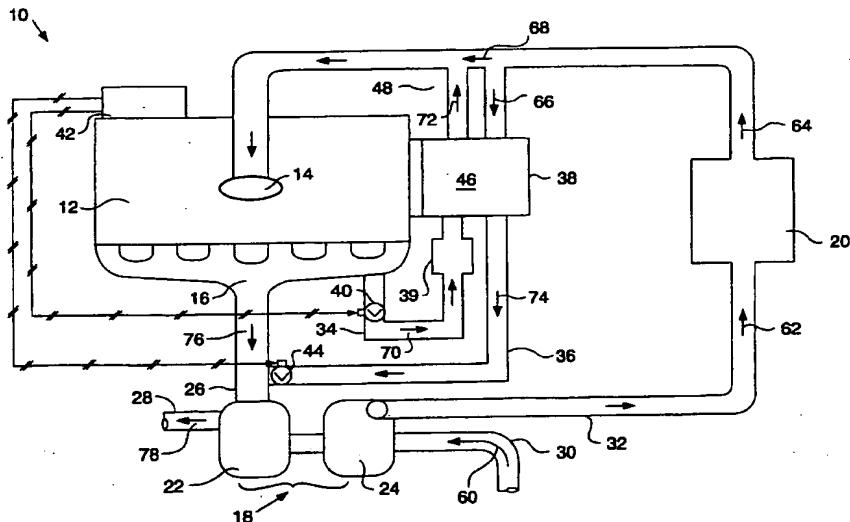
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(54) Exhaust gas recirculation system for an internal combustion engine

(57) An exhaust gas recirculation (EGR) system (10) for a turbocharged internal combustion engine (12) is provided. The EGR system (10) includes a recirculation conduit (34) for recirculating a volume of exhaust gas from the exhaust manifold (16) to the intake manifold (14); a bypass conduit (36) for diverting a flow of intake air around the engine (12); and an EGR cooler (38) disposed in operative association with the recirculation conduit (34) and the bypass conduit (36) and adapted for cooling the volume of recirculated exhaust gas in the recirculation conduit (34). The exhaust gas

recirculation cooler (38) or heat exchanger is also adapted for concurrently heating the intake air in the bypass conduit (36). The heated intake air together with the exhaust gas remaining in the exhaust manifold is used to drive the turbocharger (18). The disclosed embodiment of the exhaust gas recirculation system (10) also includes an air-to-air after-cooler (20) used to cool the compressed intake air. The cool, compressed air is then diverted to the exhaust gas recirculation cooler (38) to cool the recirculated exhaust gas.

FIG. 1



Description

The present invention relates to an exhaust gas recirculation (EGR) system for an internal combustion engine, and more particularly, to an exhaust gas recirculation cooler for use in a turbocharged diesel engine that is adapted for cooling the EGR flow and concurrently recouping some of the heat and mass flow diverted from the exhaust gas driven turbine by the EGR system.

Exhaust gas recirculation is a technique commonly used for controlling the generation of undesirable pollutant gases and particulate matter in the operation of internal combustion engines. This technique has proven particularly useful in internal combustion engines used in motor vehicles such as passenger cars, light duty trucks, and other on-road motor equipment. The exhaust gas recirculation technique primarily involves the recirculation of exhaust gas by-products into the intake air supply of the internal combustion engine. This exhaust gas thus reintroduced to the engine cylinder reduces the concentration of oxygen therein, which in turn lowers the maximum combustion temperature within the cylinder and slows the chemical reaction of the combustion process, decreasing the formation of nitrous oxide. Furthermore, the exhaust gases typically contain a portion of unburned hydrocarbon which is burned on its reintroduction into the engine cylinder, which further reduces the emission of exhaust gas by-products which would be emitted as undesirable pollutants from the internal combustion engine.

Another technique useful in the control and reduction of undesirable emissions from internal combustion engines is the use of pressure-charged intake air. This permits the use of relatively smaller cubic displacement and lighter weight internal combustion engines in mobile equipment, reducing in turn the specific fuel consumption of the vehicle and overall mass of the vehicle necessary to perform a given function. In addition to the benefits of reduced size and mass, the typical pressure-charging device may be controlled to provide improved emissions characteristics. Pressure-charging machines suitable for such applications include the exhaust gas driven turbocharger which is comprised typically of an exhaust gas driven turbine linked to a turbine disposed in the intake air stream to provide compression of the intake air. The typical turbocharger is controlled by providing a gate which controls exhaust gas flow and gates exhaust gas to bypass the exhaust gas turbine and control the charging rate of the turbocharger so that the maximum pressure limits of the associated internal combustion engine are not exceeded.

Still another technique for controlling emissions used by many engine manufacturers is the use of aftercooling the intake air thereby reducing the intake manifold temperature. Some of the related art techniques have also considered intercooling the EGR gases by routing the recirculated exhaust gases through an after-

cooler. It is well known that lower intake manifold temperatures tends to reduce the formation of nitrous oxides found in the exhaust.

When utilizing EGR in a turbocharged diesel engine, the exhaust gas to be recirculated is preferably removed upstream of the exhaust gas driven turbine associated with the turbocharger. In many EGR applications, the exhaust gas is diverted directly from the exhaust manifold. Likewise, the recirculated exhaust gas is preferably reintroduced to the intake air stream downstream of the compressor and air-to-air after-cooler. For example, in many EGR applications the recirculated exhaust gas is reintroduced to the intake manifold.

Reintroducing the exhaust gas downstream of the compressor and air-to-air aftercooler is preferred due to the reliability and maintainability concerns that arise should the exhaust gas is passed through the compressor and aftercooler. However at some engine operating conditions, there is a pressure differential between the intake manifold and the exhaust manifold which essentially prevents many conventional EGR systems from being utilized. For example, at high speed, high load conditions in a turbocharged engine, the exhaust gas does not readily flow from the exhaust manifold to the intake manifold. What is needed, therefore, is a simple and inexpensive technique for recirculating exhaust gas from the exhaust manifold to the intake manifold at all engine operating conditions.

Another problem associated with many conventional EGR systems is that the turbocharger efficiency is sacrificed when exhaust gas is diverted from the exhaust manifold. Removing the exhaust gas to be recirculated from the exhaust manifold or elsewhere upstream of the exhaust gas driven turbine depletes the mass flow and heat energy passing through the turbine which, in turn, lowers the boost levels created by the compressor. Most diesel engine turbochargers are fixed geometry turbochargers, in that they are specifically designed to operate efficiently when matched to the engine exhaust flow output. The reduction in mass flow and pressure due to the EGR creates a mismatch between the exhaust flow to the turbocharger and the turbine specifications during EGR operation. The mismatch results in a turbocharger output that is reduced in percentage more than the percentage reduction in exhaust flow to the turbocharger thereby creating significant losses in airflow and boost pressure. The reduction in airflow and boost pressure decreases the air to fuel ratio down to a point where particulates as well as the brake specific fuel consumption (BSFC) increase. Disadvantageously, the reduction in airflow and boost pressure also results in a noticeable difference in engine performance to the operator depending on whether EGR is on or off.

The present invention addresses the above and other needs by providing a method and system for exhaust gas recirculation in an internal combustion

engine, preferably a turbocharged diesel engine.

In one aspect, the invention may be characterized as an exhaust gas recirculation (EGR) system incorporating: a recirculation conduit for recirculating a volume of exhaust gas from the exhaust manifold to the intake manifold; a bypass conduit for diverting a flow of intake air around the engine; and an EGR cooler disposed in operative association with the recirculation conduit and the bypass conduit and adapted for cooling the volume of recirculated exhaust gas in the recirculation conduit. The exhaust gas recirculation cooler is also adapted for concurrently heating the intake air in the bypass conduit. The heated intake air is preferably used to replace the diverted exhaust gas. In other words, the heated intake air together with the exhaust gas remaining in the exhaust manifold are used to drive the turbine and associated compressor to pressurize the intake air.

The disclosed embodiment of the exhaust gas recirculation system also includes an air-to-air after-cooler interposed between the compressor and the intake manifold to cool the compressed intake air. The bypass conduit diverts a portion of the cool, compressed intake air to the exhaust gas recirculation cooler or heat exchanger to cool the recirculated exhaust gas.

The invention may also be characterized as a method for recirculating exhaust gas in a turbocharged internal combustion engine. The disclosed method comprises the steps of: (a) recirculating a selected volume of exhaust gas from the exhaust manifold to the intake manifold via a recirculation conduit; (b) diverting a flow of cool intake air to bypass the engine via a bypass conduit; and (c) cooling the recirculated exhaust gas in the recirculation conduit using the flow of diverted intake air. The disclosed method of recirculating exhaust gas further includes the step of heating the intake air in the bypass conduit using the recirculated exhaust gas. The step of heating the intake air in the bypass conduit and the step of cooling the recirculated exhaust gas in the recirculation conduit are preferably accomplished concurrently. The heated intake air along with any remaining exhaust gas in the exhaust manifold is then directed to the gas driven turbine of the turbocharger. In this manner, the losses in the turbocharger speed due to the exhaust gas recirculation are minimized.

Accordingly, an important aspect of the disclosed invention is the use of intake air to cool the recirculating exhaust gases. This eliminates the need to use the engine coolant to absorb the heat rejection from the recirculating exhaust gases. In addition, the use of intake air to cool the recirculating exhaust gases thus avoids or minimizes pumping losses incurred in many related art systems where jacket water flows through the EGR cooler.

Advantageously, by cooling the EGR gases to temperatures slightly above the temperature of the after-cooled intake air, the intake manifold temperatures are

lowered and the air to fuel ratio for a given volume of EGR can be increased. Lowering the intake manifold temperatures and increasing the air to fuel ratio for a given volume of EGR results in improved emissions performance.

Another important aspect of the disclosed invention is the use of the recirculated exhaust gases to heat the diverted intake air. This heated intake air is routed to the turbocharger along with the non-recirculated exhaust gases present in the exhaust manifold. Directing both the heated intake air and non-recirculated exhaust gases to the turbocharger allows the standard, fixed geometry turbochargers to operate at a more efficient temperature, pressure and air mass flow. The more efficient operation of the turbocharger during EGR operations allows for the improvement of the air to fuel ratio at high engine loads and permits the use of EGR at engine operating conditions that many related art EGR systems have avoided.

Advantageously, the present EGR system facilitates the use of EGR at high engine loads and allows for rapid transient response of a turbocharger. In other words, the turbocharger speed will remain high during EGR operation thus allowing quick response time when switching from EGR operation to non-EGR operation.

The above and other aspects, features, and advantages of the present invention will be more apparent from the following, more descriptive description thereof, presented in conjunction with the following drawings, wherein:

FIG. 1 is a schematic representation of the exhaust gas recirculation (EGR) system for a turbocharged engine in accordance with the present invention;

FIG. 2 is a schematic representation of an alternate embodiment of the exhaust gas recirculation (EGR) system for a turbocharged engine in accordance with the present invention; and

FIG. 3 is a functional block diagram of the exhaust gas recirculation (EGR) system of FIG. 1, generally depicting a detailed method for recirculating exhaust gas in a turbocharged diesel engine in accordance with the present invention.

Turning now to the drawings and particularly to FIG. 1 there is shown a schematic representation of the exhaust gas recirculation (EGR) system 10 for a turbocharged compression ignition engine 12 (i.e. diesel engine). As seen therein, the turbocharged compression ignition engine 12 includes an intake manifold 14, exhaust manifold 16, a turbocharger 18, and an air-to-air aftercooler 20. The turbocharger 18 is preferably a fixed geometry turbocharger having an exhaust gas driven turbine 22 coupled to an intake air compressor 24. The turbocharger 18 also includes an exhaust gas inlet 26 and an exhaust gas outlet 28 both in fluid communication with the exhaust gas driven turbine 22. The turbocharger 18 further includes a fresh intake air con-

duit 30 and a compressed air exit conduit 32 both of which are in fluid communication with the air compressor 24.

In the preferred embodiment, the EGR system 10 includes an EGR conduit 34, an intake air bypass conduit 36, an EGR cooler 38 or heat exchanger, and an optional particulate trap 39. As seen in FIG. 1, the EGR conduit 34 is disposed in fluid communication with the exhaust manifold 16 and is adapted for diverting a flow of exhaust gas from the exhaust manifold 16 to a position downstream of the turbocharger 18 and air-to-air aftercooler 20 and proximate the intake manifold 14. The diverted flow of exhaust gas from the exhaust manifold 16 via the EGR conduit 34 is controlled using one or more EGR valves 40 operatively associated with an engine controller 42 or similar such engine control module.

The illustrated EGR system 10 also includes the intake air bypass conduit 36 for diverting a flow of cooled, compressed intake air from a position downstream of the turbocharger 18 and air-to-air aftercooler 20 to the exhaust manifold 16. The diverted flow of cooled, compressed intake air within the bypass conduit 42 is likewise controlled using a bleed air valve 44 operating under the control of the engine controller 42.

In the illustrated embodiment, the EGR cooler 38 is a counterflow air to EGR heat exchanger. The illustrated EGR cooler 38 is adapted to receive a hot EGR input flow from the exhaust manifold 16 via the EGR conduit 34 and yield a cooled EGR output flow. The counterflow of the EGR cooler 38 is adapted to receive the diverted intake air or bleed air via bypass conduit 36. The cooled and compressed intake air is then heated by the hot EGR to produce heated intake air while simultaneously cooling the EGR flow through the EGR cooler 38. The heated intake air exiting from the EGR cooler 38 is combined with the exhaust gas remaining in the exhaust manifold 16 and used to drive the exhaust gas driven turbine 22 and associated compressor 24 thereby pressurizing the intake air approximate to the designed boost levels. As indicated above, the use of intake air to cool the EGR eliminates the need to use the engine coolant to absorb the heat from the recirculating exhaust gases and avoids pumping losses associated therewith.

Additional features of the illustrated embodiment of the EGR system include a exhaust particulate trap 39, a temperature sensor 48, and a mass flow sensor 50. The particulate trap 39, if used, is preferably disposed along the EGR conduit 34 upstream of the EGR cooler 38. Temperature sensor 48 is preferably disposed along the EGR conduit 34 proximate the intake manifold and is adapted to measure the temperature of the cooled EGR flow. Mass flow sensor 50 is located within the intake air flow circuit, preferably downstream of the turbocharger 22 and aftercooler 20 yet upstream of the location of the EGR conduit 34. The mass flow sensor is adapted to measure the non-EGR air intake to the engine. Both the

temperature sensor 48 and mass flow sensor 50 are operatively connected to the engine controller 42 where such measurements are utilized to control the operation of the EGR system 10 by controlling the bleed air valve 44 as well as the EGR valves 40.

In the embodiment illustrated in FIG. 1, the diverted exhaust gas is driven to the intake manifold 14 by the positive displacement pumping action of one or more designated cylinders. The complete diversion of exhaust gas from one or more cylinders to the EGR conduit 34 allows the EGR rate to be kept more or less constant without having to throttle the EGR valves 40. In addition, since the exhaust gas diverted from the selected cylinders is typically pressurized above that of the exhaust manifold 16 and intake manifold 14, the EGR system 10 is adapted to operate within a broader range of engine operating conditions (i.e. at high load conditions). As indicated above, there exist some engine operating conditions, such as high load conditions, where the pressure differential between the intake manifold and the exhaust manifold essentially prevents many conventional EGR systems from being utilized without expensive and inefficient throttling arrangements in either the exhaust or intake manifolds.

Turning now to FIG. 2, there is shown an alternate embodiment of the EGR system 10. In most respects, the EGR system 10 of FIG. 2 is similar to that disclosed above, with reference to FIG. 1, except for the technique and associated structure for driving the diverted exhaust gas from the exhaust manifold 16 to the intake manifold 14. In the embodiment illustrated in FIG. 2, the intake air circuit includes a venturi throat 52 disposed downstream of the air-to-air aftercooler 20 and turbocharger 18 and upstream of the intake manifold 14. The EGR conduit 34 diverts the exhaust gas from the exhaust manifold 16 to a location proximate the throat of the venturi throat 52 thereby allowing the lower pressure exhaust gas from the exhaust manifold 16 to be drawn into the intake circuit and fed into the engine 12 with the intake air. Again, since the exhaust gas diverted from the exhaust manifold 16 is typically pressurized below that of the intake manifold 14 in many engine operating conditions, the venturi throat 52 is adapted to compensate for the pressure differences and allow the EGR system 10 to operate within the a broader range of engine operating conditions (i.e. at high load conditions).

Turning again to FIG. 1, one skilled in the art can appreciate and understand the preferred method of recirculating exhaust gas associated with the illustrated embodiment. Broadly speaking, the disclosed method of recirculating exhaust gas comprises the steps of: recirculating a selected volume of exhaust gas from the exhaust manifold 16 to the intake manifold 14 via the EGR conduit 34; diverting a flow of cool intake air to bypass the engine 12 via a bypass conduit 36; and cooling the recirculated exhaust gas in the EGR conduit 34 using the flow of diverted intake air and an EGR to

intake air heat exchanger 46. The method also includes the step of concurrently heating the intake air in the bypass conduit 36 using the recirculated exhaust gas. The heated intake air is fed to the exhaust manifold 16 where it is used to replace the recirculated exhaust gas. The heated intake air is combined with the remaining exhaust gas and used to drive the turbine 22 of the turbocharger 18.

Turning now to FIG. 3, there is shown a functional block diagram generally depicting a more detailed method for recirculating exhaust gas in a turbocharged diesel engine. It is important to note that while the description hereof is present in a sequential nature, many of the actual functions involved in the preferred process are performed concurrently, and not all steps are essential to the present method.

With the foregoing in mind, the depicted method involves the steps of: (a) receiving fresh intake air 60 at the turbocharger 18; (b) compressing the fresh intake air 60 with turbocharger 18; (c) sending the compressed intake air 62 to the air-to-air aftercooler 20; (d) cooling the compressed intake air 62 using the air-to-air aftercooler 20 to yield cooled compressed intake air 64; (e) diverting a selected volume 66 of the cooled compressed intake air 64; (f) forwarding the remaining cooled compressed intake air 68 to the engine 12; and (g) measuring the mass flow of the remaining cooled compressed intake air 68 forwarded to the engine 12 using a mass flow sensor 50.

The preferred method also includes the steps of (h) diverting a selected volume of hot exhaust gas 70 from the exhaust manifold 16 of the engine 12; (i) cleansing the diverted hot exhaust gas 70 using a particulate trap 39; (j) cooling the diverted hot exhaust gas 70 to yield cooled exhaust gas 72 while concurrently heating the diverted volume of intake air 66 to yield heated intake air 74 using the heat exchanger 46; (k) measuring the temperature of the cooled exhaust gas 72 using a temperature sensor 48; (l) combining the cooled exhaust gas 72 with the cooled compressed engine intake air 68 proximate the intake manifold 14; and (m) forwarding the combined intake/EGR gas to the engine 12. Concurrently therewith the preferred method also includes: (n) replacing the hot exhaust gas 70 diverted from the exhaust manifold 16 with the heated intake air 74 and combining the heated intake air 74 with any remaining exhaust gas 76 to form a selected volume of discharge air 78; (o) driving an exhaust gas driven turbine 22 of the turbocharger 18 with the discharge air 78; and (p) forwarding the discharge air 78 to the exhaust system associated with the engine.

Claims

1. An exhaust gas recirculation system (10) in an internal combustion engine (12) having an intake circuit, an intake manifold (14) and an exhaust manifold (16), said intake circuit including an intake air
- 5 pressurizing device (18), said exhaust gas recirculation system (10) comprising:
- 10 an exhaust gas recirculation conduit (34) for diverting a flow of exhaust gas to said intake manifold (14);
- 15 an intake air bypass conduit (36) in fluid communication with said intake circuit for permitting a flow of intake air from said intake circuit; and an exhaust gas recirculation cooler (38) disposed in operative association with said exhaust gas recirculation conduit and said intake air bypass conduit (36) and adapted for cooling said exhaust gas in said exhaust gas recirculation conduit (34).
- 20 2. The exhaust gas recirculation system (10) of claim 1 wherein said exhaust gas recirculation cooler (38) is further adapted for heating said intake air in said intake air bypass conduit (36).
- 25 3. The exhaust gas recirculation system (10) of claim 2 wherein said air intake pressurizing device (18) includes a compressor (24) and a gas driven turbine (22), said gas driven turbine (22) adapted to receive exhaust gas from said exhaust manifold (16) together with said heated intake air from said air intake bypass conduit (36) and drive said compressor (24) thereby pressurizing said intake air in said intake circuit.
- 30 4. The exhaust gas recirculation system (10) of claim 3 wherein said intake circuit further includes an air-to-air after-cooler (20) interposed between said intake manifold (14) and said compressor (24) and in fluid communication therewith to cool said compressed intake air.
- 35 5. The exhaust gas recirculation system (10) of claim 4 wherein said intake air bypass conduit (36) is disposed in fluid communication with said intake circuit at a location downstream of said air-to-air after-cooler (20), said intake air bypass conduit (36) adapted for permitting a flow of intake air from said intake circuit to said exhaust manifold (16).
- 40 6. The exhaust gas recirculation system (10) of claim 1 wherein said exhaust gas recirculation cooler (38) is a heat exchanger for transferring heat from said recirculated exhaust gas to said intake air in said intake air bypass conduit (36) thereby cooling said recirculated exhaust gas and concurrently heating said intake air in said intake air bypass conduit (36).
- 45 7. The exhaust gas recirculation system (10) of claim 1 wherein said exhaust gas recirculation conduit (34) is disposed in fluid communication with said exhaust manifold (16) for permitting a flow of

exhaust gas from said exhaust manifold (16) to said intake manifold (14).

8. The exhaust gas recirculation system (10) of claim 1 wherein said exhaust gas recirculation conduit (34) is disposed in fluid communication with one or more cylinders for driving a flow of exhaust gas from said cylinders to said intake manifold (14). 5

9. The exhaust gas recirculation system of claim 1 wherein said intake circuit further includes a venturi throat (52) disposed downstream of said air intake pressurizing device (18), and wherein said exhaust gas recirculation conduit (34) is disposed in fluid communication with said venturi throat (52) for driving a flow of exhaust gas to said intake manifold (14) via said venturi throat (52). 10

10. The exhaust gas recirculation system (10) of claim 1 further including a particulate trap (39) disposed in operative association with said exhaust gas recirculation conduit (34) for cleansing said recirculating exhaust gas. 20

11. The exhaust gas recirculation system (10) of claim 1 further including one or more valves (40) for controlling the flow of recirculated exhaust gas in said exhaust gas recirculation conduit (34) and the flow of diverted intake air in said intake air bypass conduit (36). 25

12. The exhaust gas recirculation system (10) of claim 11 further including a temperature sensor (48) disposed along said exhaust gas recirculation conduit (34) and adapted for measuring the temperature of said cooled recirculated exhaust gas and controlling said valves (40) in response thereto. 30

13. The exhaust gas recirculation system (10) of claim 11 further including a mass flow sensor (50) disposed along said intake circuit and adapted for measuring the mass flow rate of said compressed intake air not diverted to said intake air bypass conduit (36) and controlling said valves (40) in response thereto. 35

14. A method of recirculating exhaust gas in an internal combustion engine (12), said engine (12) having an intake circuit including an intake air pressurizing device (22), an intake manifold (14), and an exhaust manifold (16), the method comprising the steps of: 50

- (a) recirculating a selected volume of exhaust gas to said intake manifold (14) via a recirculation conduit (34);
- (b) diverting a flow of intake air from said intake circuit via a bypass conduit (36); and
- (c) cooling said recirculated exhaust gas in said

recirculation conduit (34) using said flow of diverted intake air. 55

15. The method of recirculating exhaust gas as set forth in claim 14 further comprising the step of heating said intake air in said bypass conduit using said recirculated exhaust gas, the step of heating said intake air being concurrent with the step of cooling said recirculated exhaust gas.

16. The method of recirculating exhaust gas as set forth in claim 15 wherein said air intake pressurizing device (22) includes a gas driven turbine (22) and a compressor (24), said method further includes the step of driving said gas driven turbine (22) with any exhaust gas remaining in said exhaust manifold (16) together with said heated intake air from said air intake bypass conduit (36) which in turn drives said compressor (24) thereby pressurizing said intake air in said intake circuit. 15

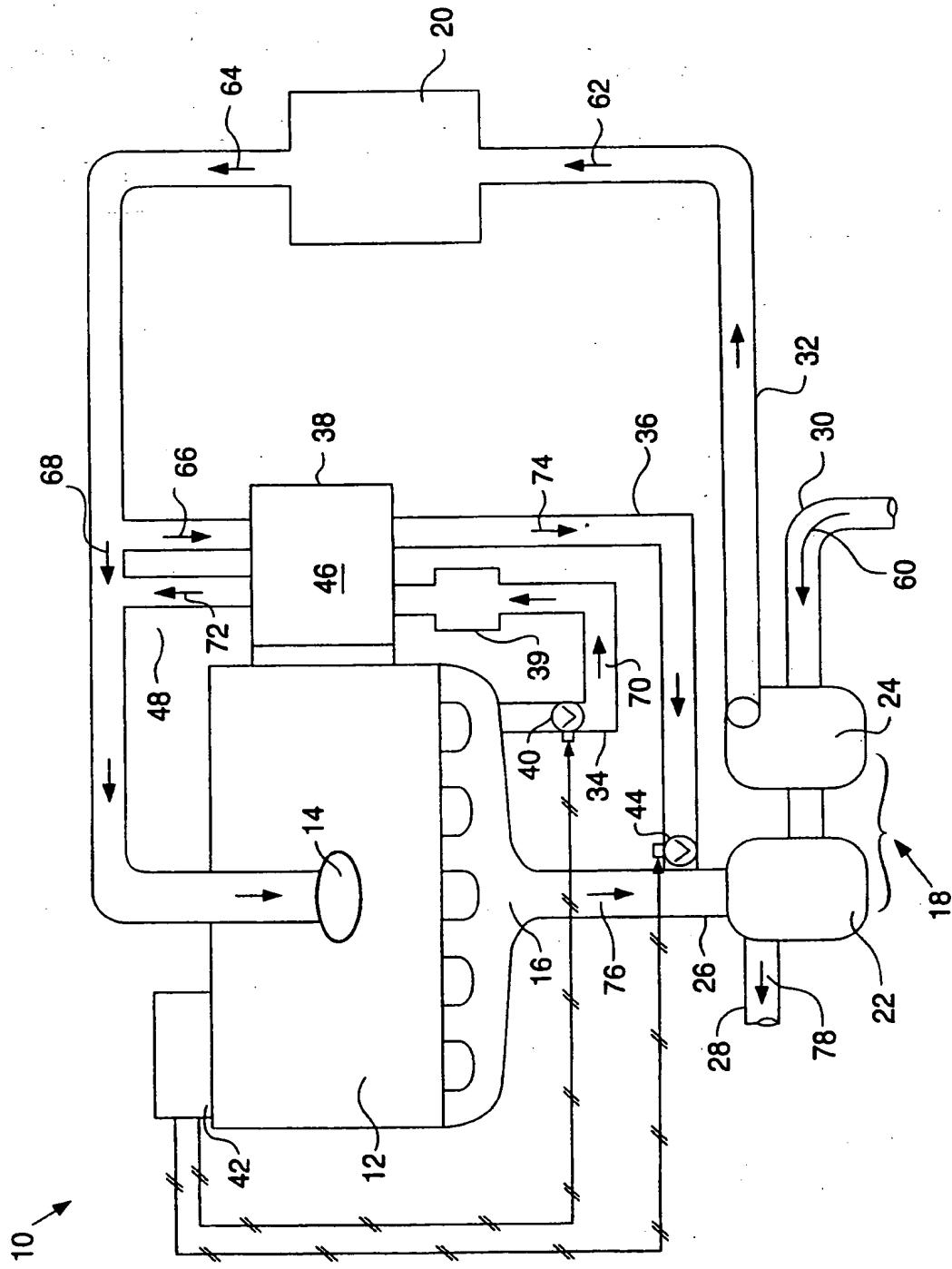
17. The method of recirculating exhaust gas as set forth in claim 16 further comprising the step of cooling said compressed intake air in said intake circuit prior to diverting said flow of intake air from said intake circuit via said bypass conduit. 20

18. The method of recirculating exhaust gas as set forth in claim 14 further comprising the step of cleansing said recirculated exhaust gas prior to said cooling step. 25

19. The method of recirculating exhaust gas as set forth in claim 14 further comprising the step of measuring the mass flow of said compressed intake air not diverted and controlling a volume of intake air diverted and volume of exhaust gas recirculated in response thereto. 30

20. The method of recirculating exhaust gas as set forth in claim 14 further comprising the step of measuring the temperature of said cooled recirculated exhaust gas and controlling a volume of intake air diverted and volume of exhaust gas recirculated in response thereto. 35

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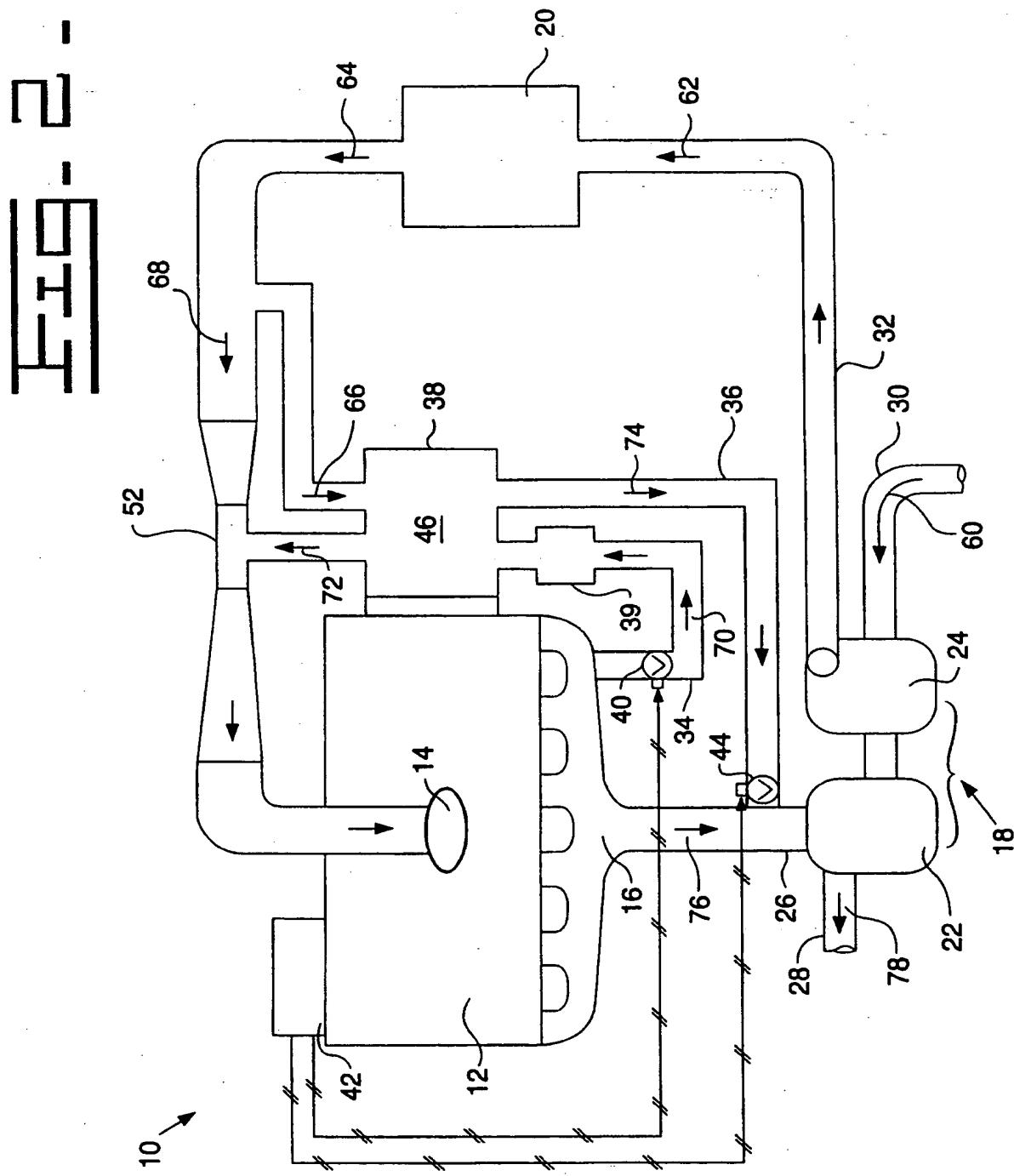
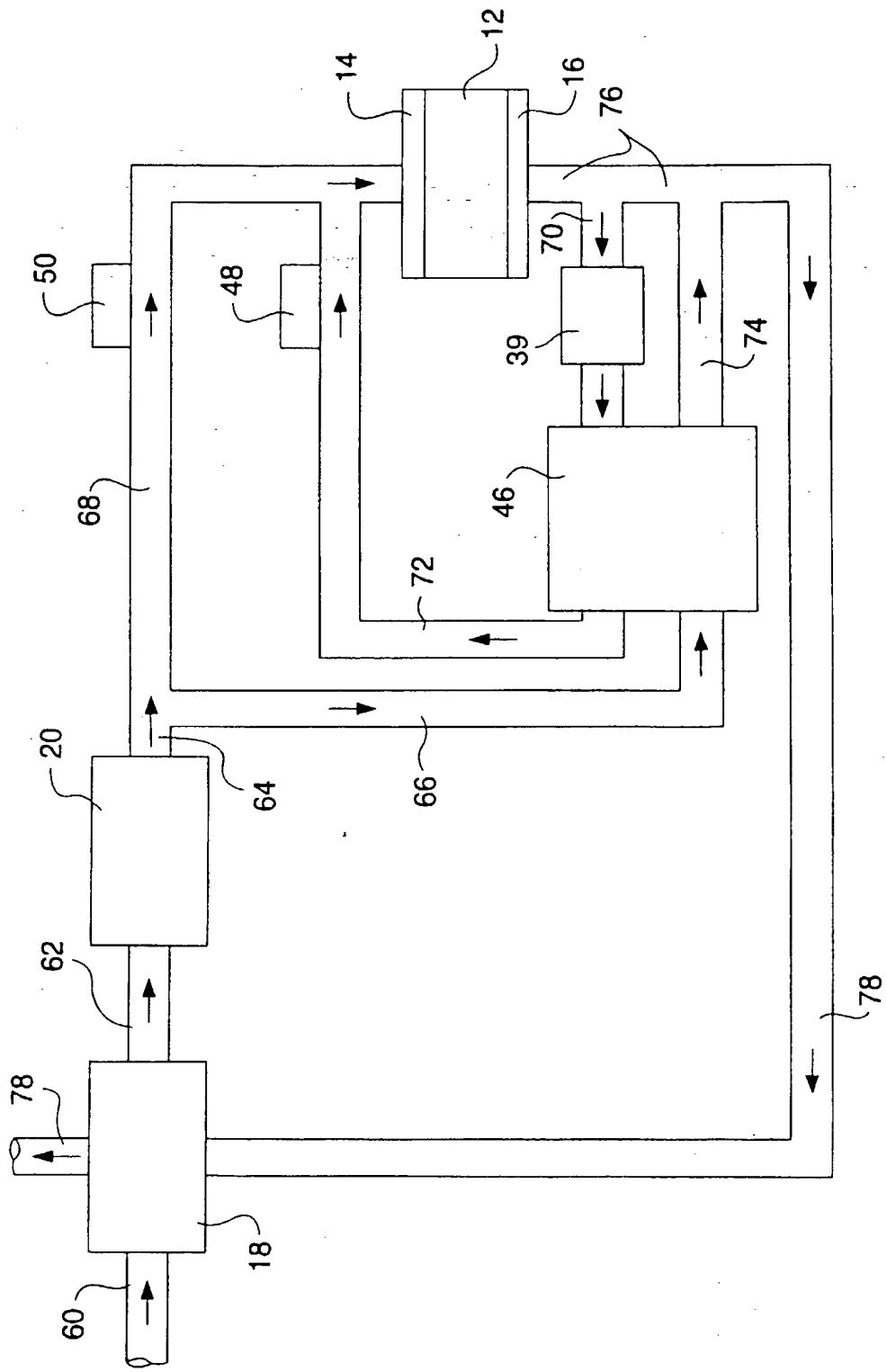


FIG. 3







**European Patent
Office**

EUROPEAN SEARCH REPORT

Application Number

DOCUMENTS CONSIDERED TO BE RELEVANT					
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int.Cl.6)		
Y	DE 43 19 380 A (AVL GESELLSCHAFT FÜR VERBRENNUNGSKRAFTMASCHINEN UND MESSTECHNIK) 16 December 1993 * abstract * * column 2, line 21 - line 52; figure 1 * ---	1, 2, 6, 7, 9, 11, 14, 15 3, 4, 16, 17	F02M25/07 F02B29/04 F02B37/16		
A	DE 44 22 966 C (MTU MOTOREN- UND TURBINEN-UNION FRIEDRICHSHAFEN) 18 May 1995 * abstract * * column 2, line 8 - line 42; figure 1 * ---	1, 2, 6, 7, 9, 11, 14, 15			
A	DE 43 31 509 C (MTU MOTOREN- UND TURBINEN-UNION FRIEDRICHSHAFEN) 21 July 1994 * column 3, line 16 - line 37; figures 1,2 * ---	8, 10, 18			
A	US 5 440 880 A (CEYNOW ET AL.) 15 August 1995 * column 2, line 65 - column 3, line 7; figure 1 * ---	12, 13, 19, 20			
A	US 5 611 203 A (HENDERSON ET AL.) 18 March 1997 * abstract; figure 3 * ---	1, 14	F02M F02B		
A	DE 42 40 239 A (SCHMITZ) 9 June 1994 * column 3, line 38 - column 4, line 35; figure 1 * -----	1, 14			
The present search report has been drawn up for all claims					
Place of search	Date of completion of the search	Examiner			
THE HAGUE	14 July 1998	Van Zoest, A			
CATEGORY OF CITED DOCUMENTS					
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